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# Duct Silencer Modeling With COMSOL Multiphysics®

## *COMSOL Conference Online Event*

October 14 – 15, 2020

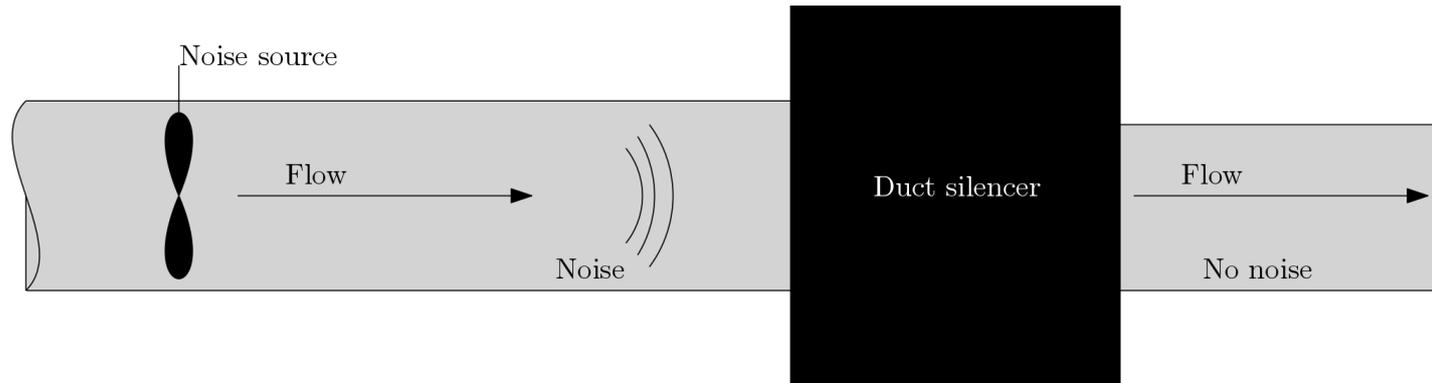
Dr.ir. Anne de Jong



acoustics | software | consultancy | engineering | education

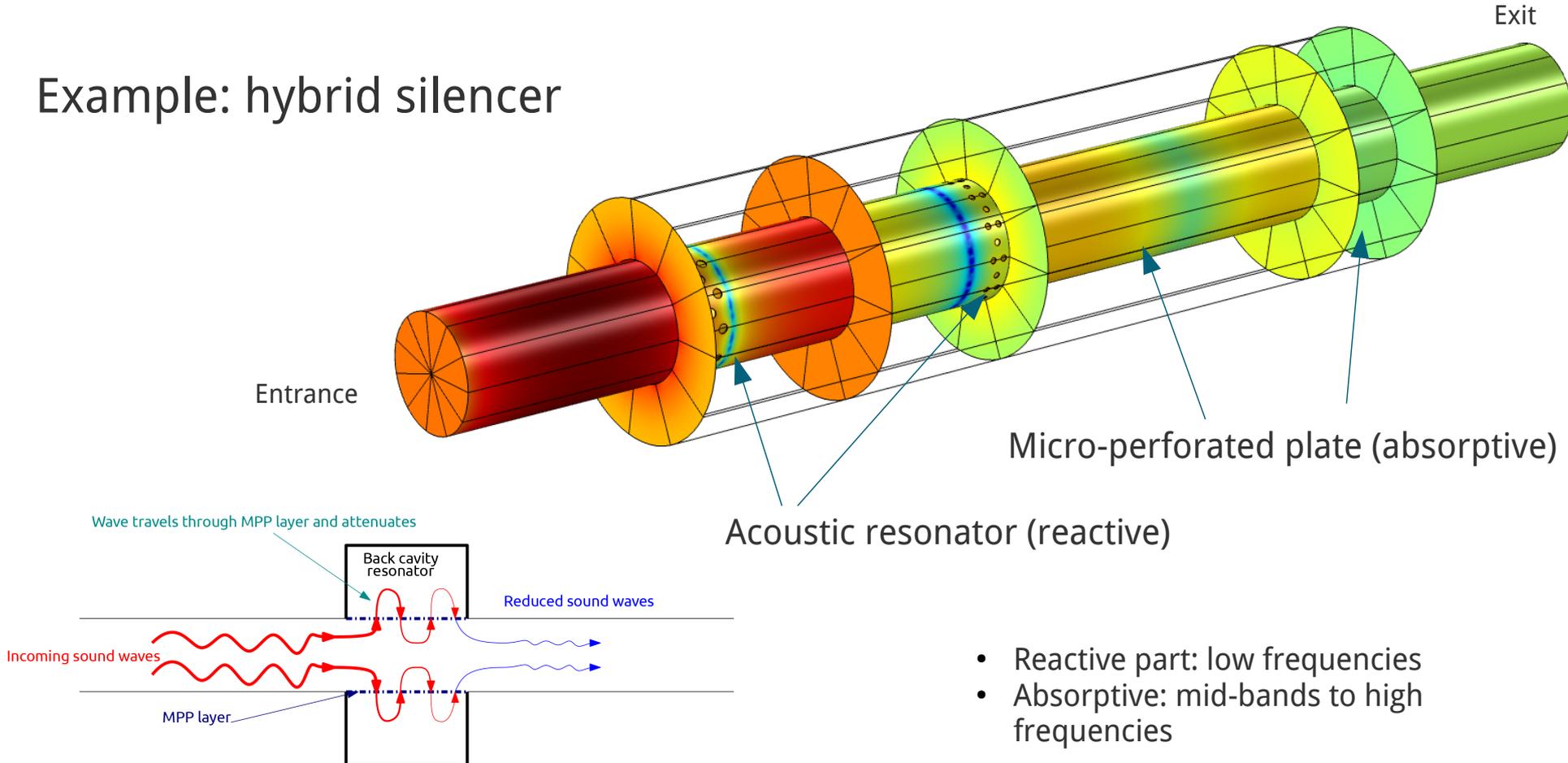
# Introduction

- Duct acoustic silencing
  - Applications:
    - Exhaust gas streams
    - Heating, Ventilation, Air Conditioning



# Duct silencer internals

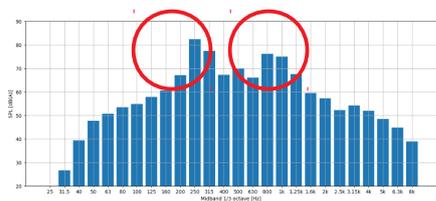
Example: hybrid silencer



# The approach of ASCEE

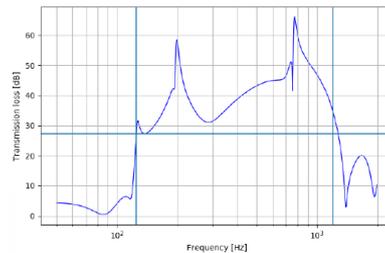
- We design custom silencers
- Benefits:
  - Optimized result by tuning duct silencer to source
  - Free flow-through, no flow head losses (pressure drop)
  - Smallest silencer for a required noise reduction

Source spectrum (noise problem)



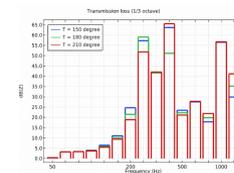
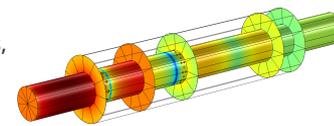
Required reduction  
(15 dB(A))

1D optimizer (Transfer matrix method + genetic algorithm)



Chamber volumes,  
Resonance frequencies,  
Lengths, etc

COMSOL Modeling



Final dimensions

Design & Construction

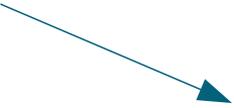


# Limitations of transfer matrix method

- Local near fields of resonators interact with each other (evanescent wave coupling)
- Only valid for plane waves (below cut-on frequency only):

$$f \ll f_c = \frac{c_0}{1.7D}$$

- Resonator hole dimension fine-tuning
  - First step: hole models (often in axisymmetric case)
  - Second step: actual holes



Interior Perforated Plate

Model type: Thin plate

Hole diameter:  $d_h$  1 [mm]

Plate thickness:  $t_p$  1.5 [mm]

Area porosity:  $\sigma$  0.1

End correction: Built in

Hole-hole interaction: Fok function

Discharge coefficient (linear):  $C_D^{(lin)}$  1

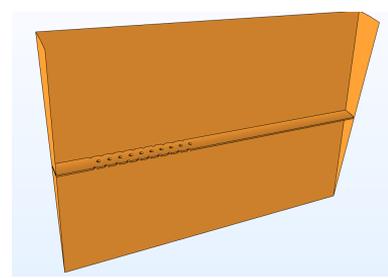
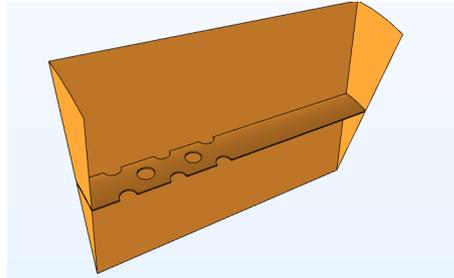
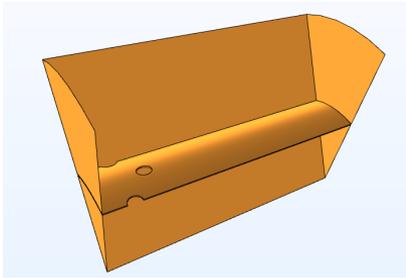
Include nonlinear effects

User-defined contribution

# COMSOL Steps (1): Resonator fine-tuning

- Real-hole geometry (no perforate modeling)
  - Comsol parts library, parametrized resonator chamber

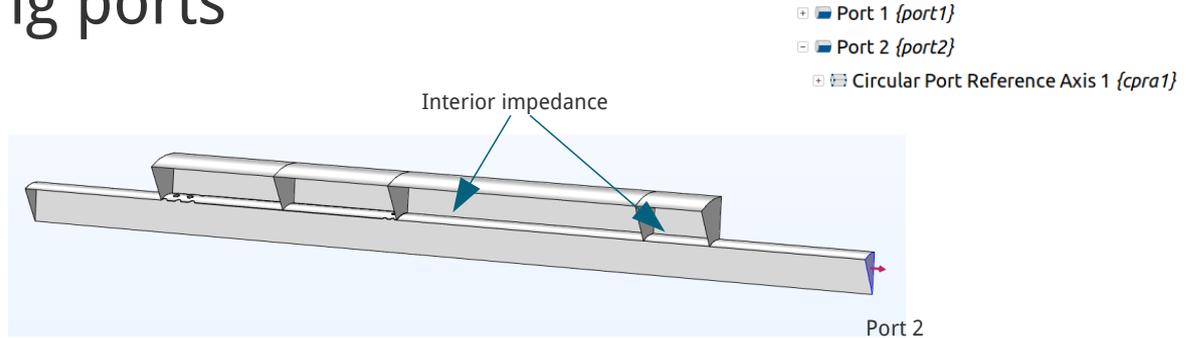
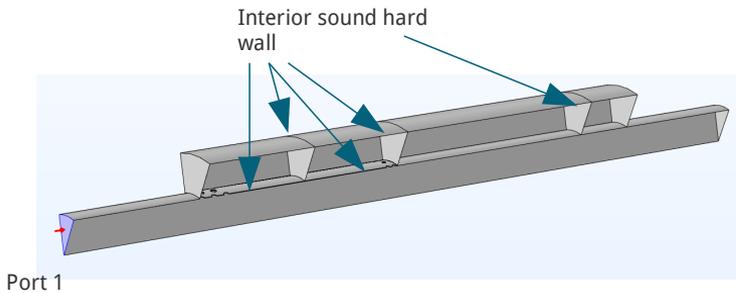
- ▢  Geometry Parts
  - ▢  Round silencer parts {grp1}
    - +  Helmholtz\_chamber\_round {part1}
    - +  entrance\_exit\_slice {part3}
    - +  Concentric MPP chamber {part4}
    - +  MPP Cavity Helmholtz slice {part5}



- These resonator chambers all come from the same library part
- Selections are used for more easy application of boundary conditions
- Axi-symmetric slice size is automatically computed

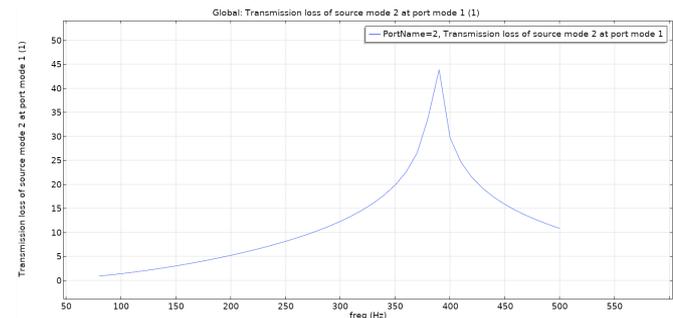
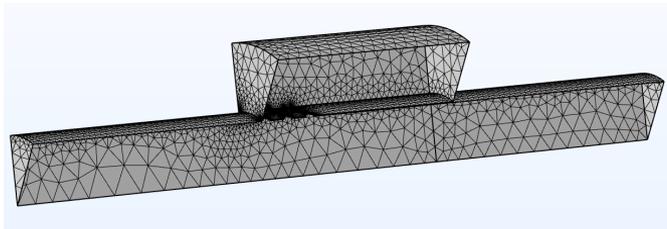
# Computation of transmission loss

- Acoustics library: using ports



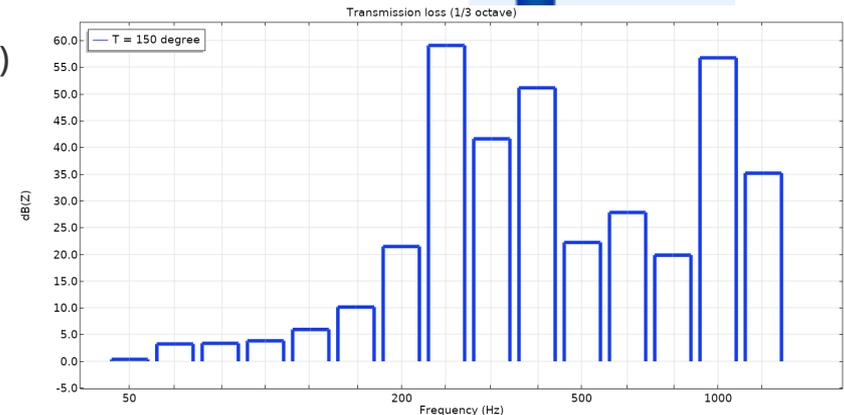
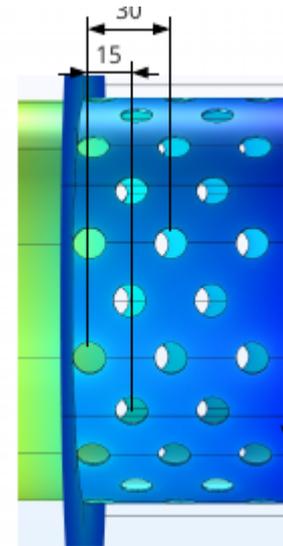
- Tune individual chambers first

- Holes: reactance is already in Helmholtz equation. Viscous resistance term added to holes from global equations, using an interior impedance boundary condition



# Assembling all together

- Note: transmission loss (TL) is a silencer property, while the final result the client is often interested in, is the insertion loss (IL).
  - TL: The ratio between the sound power of the incoming wave and the sound power of the transmitted wave without any reflections from the downstream end of the silencer.
  - IL: the difference in level prior to insertion of the silencer minus the level after inserting a silencer
- IL is hard to compute, as it depends on the exact installation, pipe lengths, exit radiation and source impedance.
  - **This analysis is done by exporting the derived acoustic transfer matrix back into the transfer matrix 1D code.**
- However, a high TL mostly results in high IL (there are exceptions)



# Conclusions

- We built a streamlined silencer design methodology for duct acoustic problems, involving
  - Source spectra tuning
  - 1D Optimal (reactive / hybrid) silencer design, without fiber-containing materials)
  - Further tuning in COMSOL for final design
  
- We have become quite effective and successful with this technology and methodology
  
- I am looking forward to answering your boiling questions!

